

— QUANTUM HEAT ENGINES AND REFRIGERATORS —

Michele Campisi

Scuola Normale Superiore, Pisa, Italy



COLLABORATIONS

Rosario Fazio, ICTP Trieste
SNS Pisa

Jukka Pekola, Aalto, Helsinki

Francesco Giazotto

Pauli Virtanen

Giampiero Marchegiani

NEST, Pisa.

OUTLINE

① Heat engines as driven bi-partite systems : SWAP ENGINE

M. Campisi, JPA 47 24 1 (2014)

M. Campisi, R. Fazio, J. Pekola, NJP 17 035012 (2014)

M. Campisi, R. Fazio, JPA 49 345002 (2016)

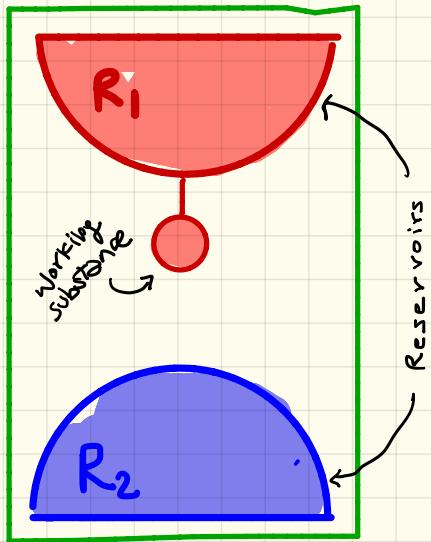
② The Power of a critical heat engine

M. Campisi, R. Fazio, Nature Communications 7 11895 (2016)

③ Josephson quantum heat engine

G. Marchegiani, P. Virtanen, F. Giavotto, M. Campisi, Phys. Rev. Applied 6 054014 (2016)

HEAT ENGINE AS A DRIVEN BI-PARTITE SYSTEM



$$\text{Subsystem 1} = R_1 + \text{WS}$$

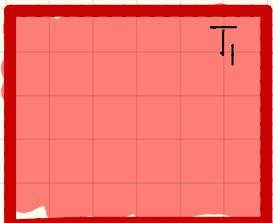
$$\text{Subsystem 2} = R_2$$

Driving = Operations on WS
and switch ON/OFF couplings

Examples: 4 Stroke engines (Carnot, Diesel, etc)

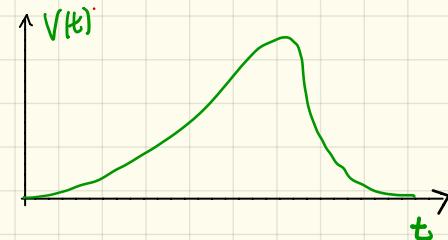
Continuous engines

FLUCTUATION RELATION FOR HEAT ENGINES



$$H(t) = H_1 + H_2 + V(t) \leftarrow \begin{matrix} \text{Work} \\ \text{Source} \end{matrix}$$

$$\rho_0 = \frac{e^{-\beta_1 H_1}}{Z_1} \otimes \frac{e^{-\beta_2 H_2}}{Z_2}$$



$$t=0$$

$$E_{n_1}^1$$

$$E_{n_2}^2$$

$$P_{n_1 n_2} = \frac{e^{-\beta_1 E_{n_1}^1}}{Z_1} \frac{e^{-\beta_2 E_{n_2}^2}}{Z_2}$$

$$t=\tau$$

$$E_{m_1}^1$$

$$E_{m_2}^2$$

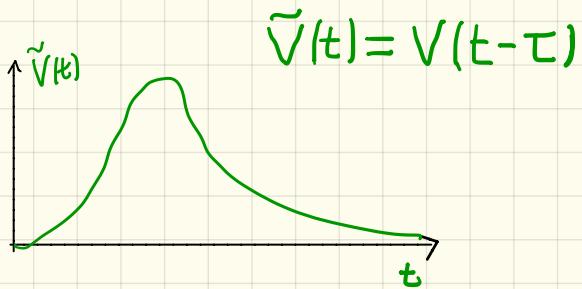
$$P_{m_1 m_2 | n_1 n_2} = |\langle m_1 m_2 | U | n_1 n_2 \rangle|^2$$

$$\downarrow$$

$$\downarrow$$

$$\Delta E_1 + \Delta E_2 = \omega$$

$$P(\Delta E_1, \Delta E_2) = \sum_{\substack{n_1, n_2 \\ m_1, m_2}} P_{n_1, n_2} P_{m_1, m_2 | n_1, n_2} \delta[\Delta E_1 - (E'_{m_1} - E'_{n_1})] \times \\ \times \delta[\Delta E_2 - (E^2_{m_2} - E^2_{n_2})]$$



$$\tilde{P}_{m_1, m_2 | n_1, n_2} = P_{n_1, n_2 | m_1, m_2}$$



$$\frac{P(\Delta E_1, \Delta E_2)}{\tilde{P}(-\Delta E_1, -\Delta E_2)} = e^{+\beta_1 \Delta E_1} e^{+\beta_2 \Delta E_2}$$

Change of variables $W = \Delta E_1 + \Delta E_2$

$$\frac{P(\Delta E_1, W)}{\tilde{P}(-\Delta E_1, -W)} = e^{+(\beta_1 - \beta_2)\Delta E_1 + \beta_2 W}$$

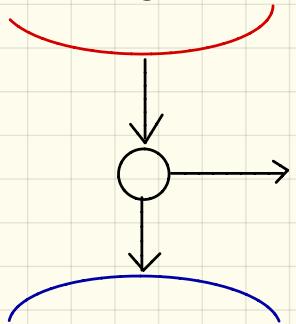
HEAT ENGINE FLUCTUATION
RELATION

$$\Rightarrow \langle e^{-(\beta_1 - \beta_2)\Delta E_1 - \beta_2 W} \rangle = 1$$

$$\Rightarrow (\beta_1 - \beta_2) \langle \Delta E_1 \rangle - \beta_2 \langle W \rangle \geq 0$$

Sinitsyn JPA 44 405001 (2011)
 Campisi JPA 47 245001 (2019)
 Campisi et al. NJP 17 035012 (2014)

Heat engine operation



$$\langle W \rangle \leq 0$$

$$\langle \Delta E_1 \rangle \leq 0$$

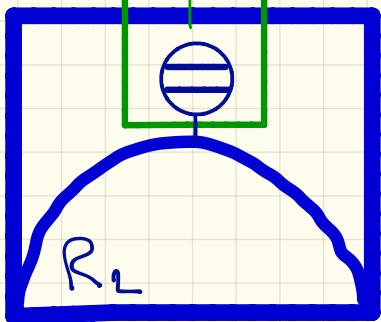
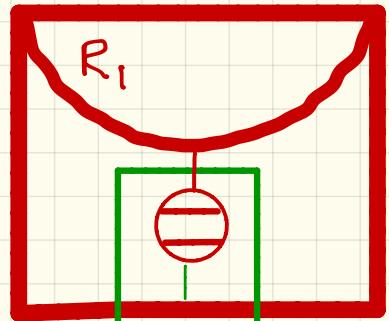
$$(\beta_1 - \beta_2) \langle \Delta E_1 \rangle - \beta_2 \langle W \rangle \geq 0$$

$$\Rightarrow \eta \doteq \frac{\langle W \rangle}{\langle \Delta E_1 \rangle} \leq 1 - \frac{\beta_2}{\beta_1} \doteq \eta^c$$

Fluctuation Theorem \Rightarrow

$$\eta \leq \eta^c$$

EXAMPLE : TWO QUBIT ENGINE



SIMPLIFYING

ASSUMPTION: Driving fast compared to
thermal relaxation

$$\frac{w_1}{2} \sigma_1^z \quad \frac{w_2}{2} \sigma_2^z$$

$$H = H_1 + H_2 + V(t) \rightarrow \cup$$

$$\langle \Delta E_1 \rangle = \text{Tr}_1 \text{Tr}_2 H_1 (U \rho U^\dagger - \rho)$$

+

$$\langle \Delta E_2 \rangle = \text{Tr}_1 \text{Tr}_2 H_2 (U \rho U^\dagger - \rho)$$

=

$$\langle W \rangle$$

MAXIMIZATION OF WORK OUTPUT

$$U = \begin{pmatrix} e^{i\varphi_1} & & & \\ & \hline & & \\ & 0 & e^{i\varphi_3} & \\ & \hline & & \\ e^{i\varphi_2} & & 0 & \\ & \hline & & e^{i\varphi_4} \end{pmatrix}$$

\leftarrow SWAP

Basis: $|++\rangle, |+-\rangle, |-+\rangle, |--\rangle$

$$|+-\rangle = e^{i\varphi_3} |-+\rangle$$

$$|-+\rangle = e^{i\varphi_2} |+-\rangle$$

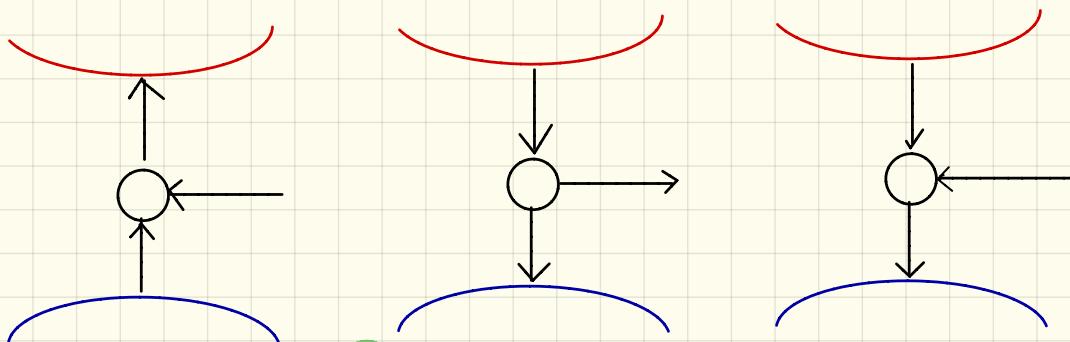
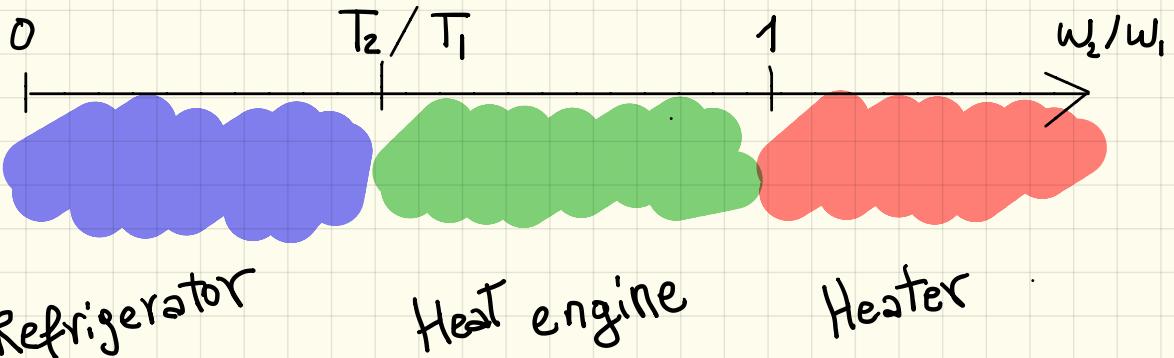
$$\langle \Delta E_1 \rangle = - \left(\frac{1}{1 + e^{\beta_1 w_1}} - \frac{1}{1 + e^{\beta_2 w_2}} \right) w_1$$

$$\langle \Delta E_2 \rangle = \left(\frac{1}{1 + e^{\beta_1 w_1}} - \frac{1}{1 + e^{\beta_2 w_2}} \right) w_2$$

$$\langle W \rangle = \left(\frac{1}{1 + e^{\beta_1 w_1}} - \frac{1}{1 + e^{\beta_2 w_2}} \right) (w_2 - w_1)$$

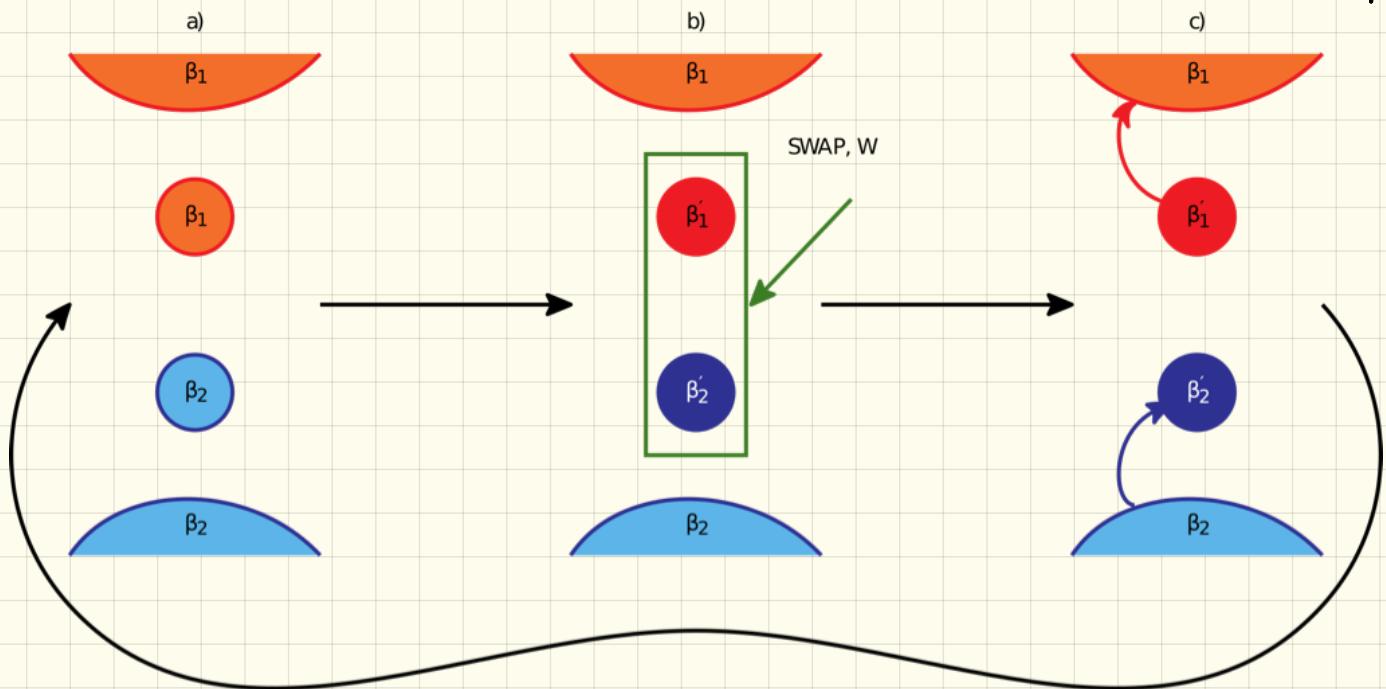
Allahverdyan et al.

PRE 77 09/18 (2064)



$$\eta = 1 - \frac{w_2}{w_1} \leq 1 - \frac{T_2}{T_1}$$

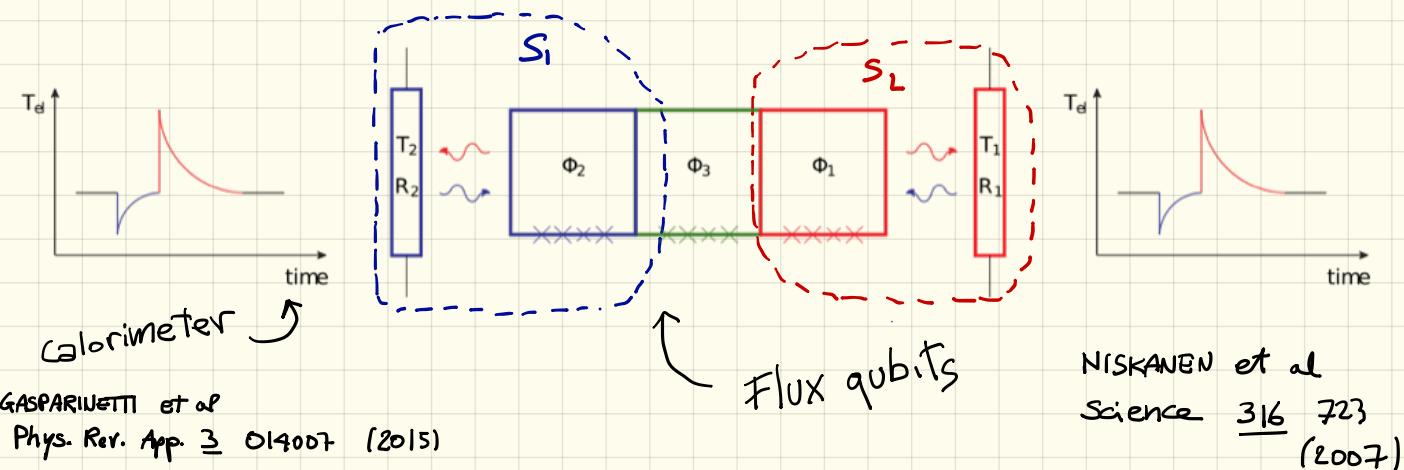
Refrigerator

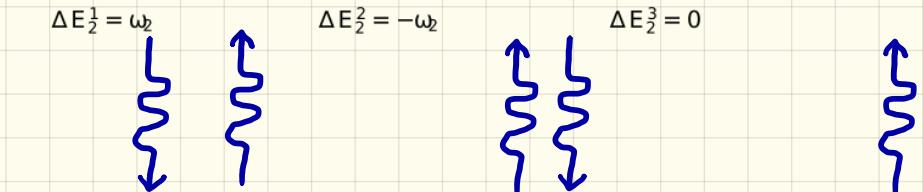
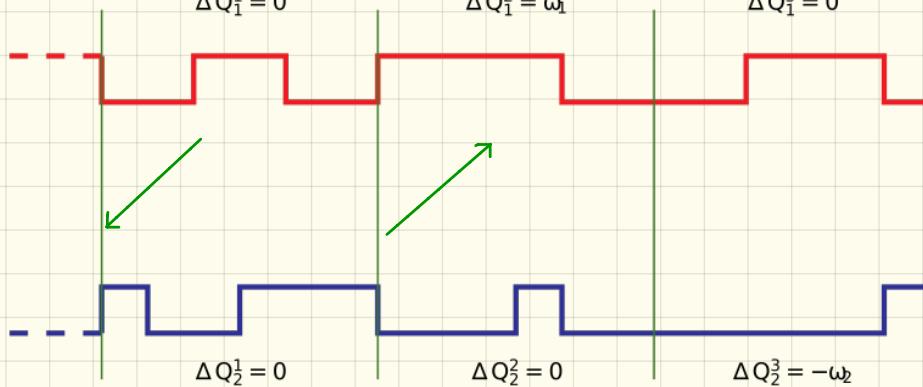
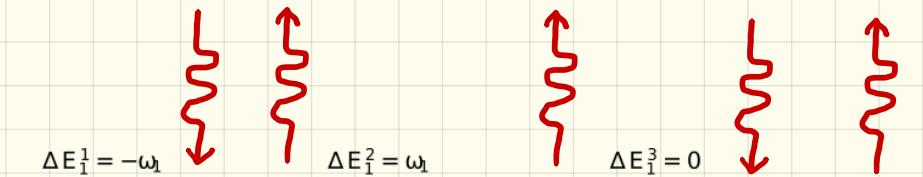


$$\beta'_1 = \beta_2 \frac{w_2}{w_1}$$

$$\beta'_2 = \beta_1 \frac{w_1}{w_2}$$

IMPLEMENTATION

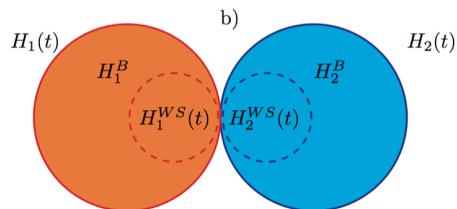
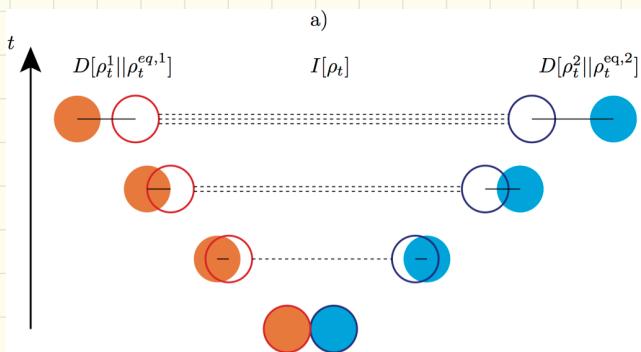




$$\Delta E_2 = -\frac{\omega_2}{\omega_1} \Delta E_1$$

$$\frac{W}{\Delta E_1} = 1 - \frac{\omega_2}{\omega_1}$$

DISSIPATION, CORRELATIONS AND LAGS IN HEAT ENGINES



$$\rho_0 = \rho_0^1 \otimes \rho_0^2 = \frac{e^{-\beta_1 H_1/10}}{z_1} \otimes \frac{e^{-\beta_2 H_2/10}}{z_2}$$

$$\beta_1 W_{diss}^{(1)} + \beta_2 W_{diss}^{(2)} = D[\rho_t^1 || \rho_t^{eq,1}] + D[\rho_t^2 || \rho_t^{eq,2}] + I_{1/2}[\rho_t]$$

$$D[\rho || \sigma] = \text{Tr } \rho \ln \rho - \text{Tr } \rho \ln \sigma$$

$$I_{1/2}[\rho] = \sum_i S[\rho^i] - S[\rho]$$

$$S[\rho] = -\text{Tr } \rho \ln \rho$$

$$W_{diss}^{(i)} = \text{Tr}_i [H_i(t)\rho_i^i - H_i(0)\rho_0^i] - \Delta F_i$$

$$F_i = -\frac{1}{\beta_i} \ln z_i$$

For heat engines $\Delta F_i = 0$

$$W_{\text{diss}}^{(i)} = \text{Tr}_i [H_i(\tau) g_{\tau}^i - H_i(0) g_0^i] = \langle \Delta E_i \rangle$$

$$\beta_{\text{MAX}} W_{\text{out}} \delta\eta \geq D[S_t^1 \| S_t^{e_{q,1}}] + D[S_t^2 \| S_t^{e_{q,2}}] + I_{1/2}[S_t] \geq 0$$

$$W_{\text{out}} \geq 0$$

$$\delta\eta = \frac{\eta^c}{\eta} - 1$$

$$\eta^c = 1 - \frac{\beta_{\min}}{\beta_{\max}}$$

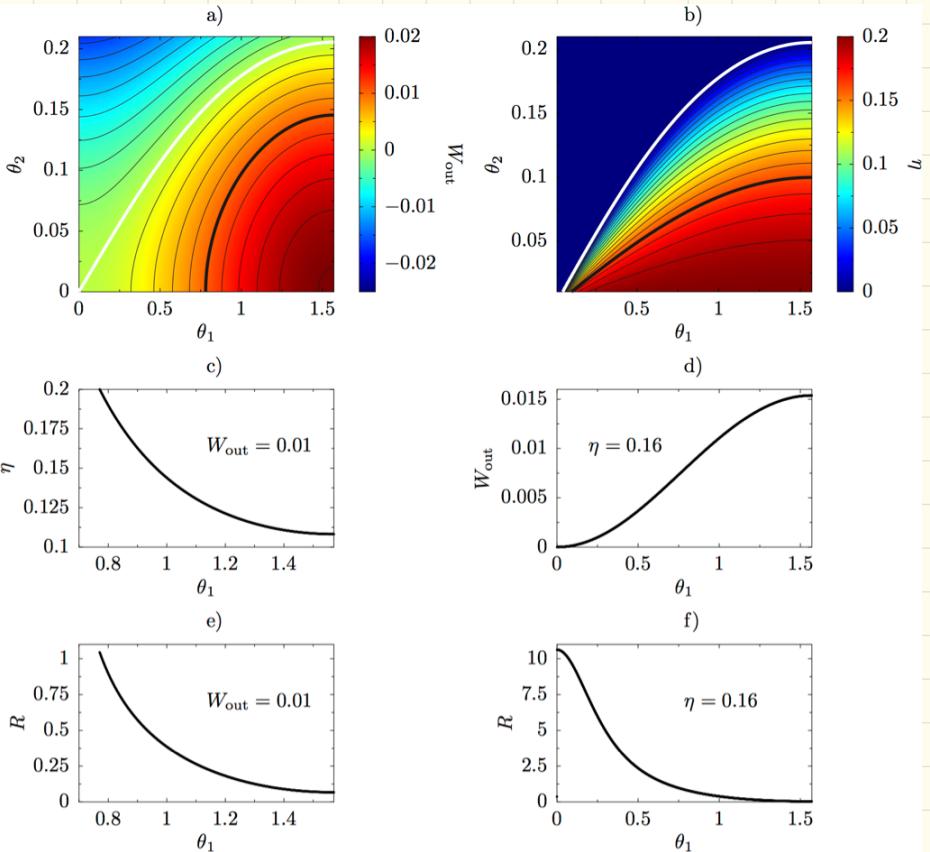
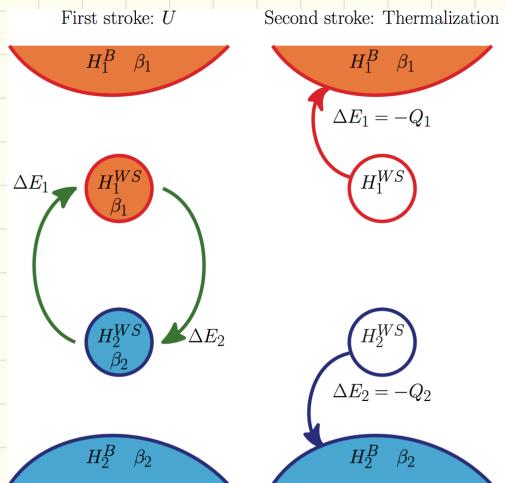
$$\eta \leq \eta^c$$

$$W_{\text{out}} = 0 \quad @ \quad \eta = \eta^c$$

$$H(t) = \frac{\omega_1}{2} \sigma_z' + \frac{\omega_2}{2} \sigma_z^2 + V(t)$$

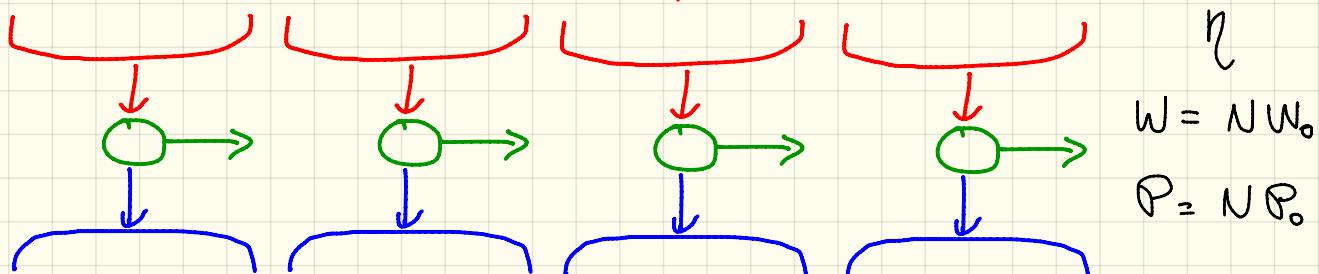
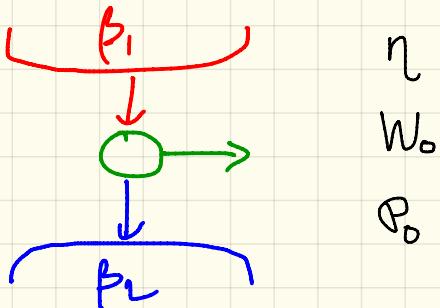
$$U = U_{\text{swap}}(\theta_1) \cdot U_{\text{cnot}}(\theta_2)$$

$$R[\rho_\varepsilon] = \frac{I[\rho_\varepsilon]}{D[\rho_\varepsilon' \parallel \rho_\varepsilon^{\text{eq},1}] + D[\rho_\varepsilon^2 \parallel \rho_\varepsilon^{\text{eq},2}]}$$

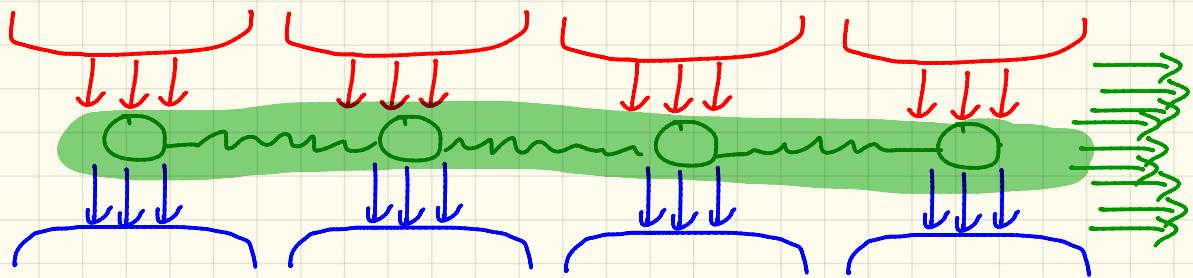


The power of a
critical heat engine

Increase power without affecting efficiency



Increasing Power AND efficiency via interactions?



Performance

$$\Pi = \frac{w}{\Delta \eta}$$

Performance
rate

$$\hat{\Pi} = \frac{P}{\Delta \eta}$$

$$\Delta \eta = \eta^c - \eta$$

Parallel Device

$$w \sim N$$

$$\Delta \eta \sim 1 \Rightarrow \Pi \sim N$$

$$P \sim N$$

$$\Delta \eta \sim 1$$

$$\hat{\Pi} \sim N$$

- Can interactions boost performance $\dot{\Pi}$?
- Can interactions boost performance rate $\ddot{\Pi}$?

Boosted performance rate \Rightarrow approach to Carnot efficiency

@ fixed power per resource

if $\dot{\Pi} = \frac{P}{\Delta\eta} \sim N^{1+\alpha}$ ($\alpha > 0$)

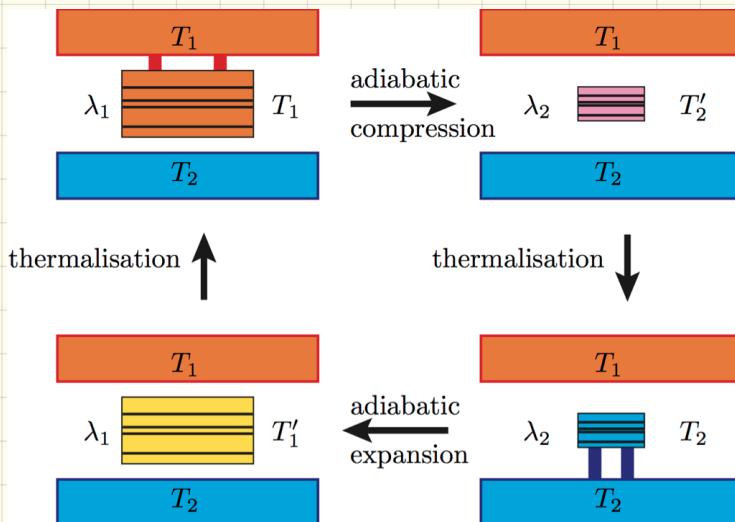
then $\Theta \sim N$ with $\Delta\eta \sim \frac{1}{N^\alpha} \rightarrow 0$ is possible

\rightarrow one can asymptotically approach $\Delta\eta \rightarrow 0$ i.e. $\eta \rightarrow \eta^c$

"Quantum" Otto engine

Working substance

$$H^{ws}(t) = \lambda(t) K$$



$$\overline{T}_1^{-1} = \overline{T}_2 \frac{\lambda_1}{\lambda_2}$$

$$\overline{T}_2^{-1} = \overline{T}_1 \frac{\lambda_2}{\lambda_1}$$

$$Q_i = \mp \lambda_i \left[U_K \left(\beta_2 \lambda_2 \right) - U_K \left(\beta_1 \lambda_1 \right) \right]$$

$$W_{\text{out}} = Q_1 + Q_2$$

$$= (\lambda_2 - \lambda_1) \left[U_K \left(\beta_2 \lambda_2 \right) - U_K \left(\beta_1 \lambda_1 \right) \right]$$

$$\left(U_K(\theta) = \frac{\text{Tr } K e^{-\theta K}}{\text{Tr } e^{-\theta K}} \right)$$

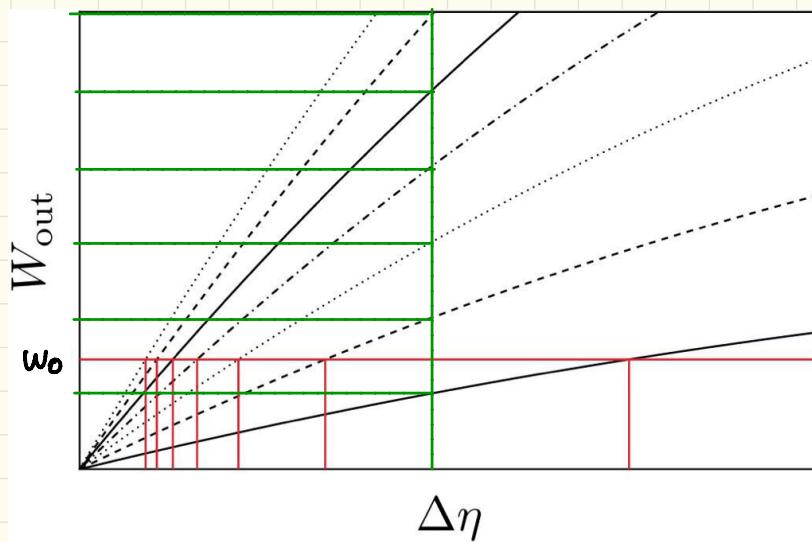
$$\eta = \frac{W_{\text{out}}}{Q_1} = 1 - \frac{\lambda_2}{\lambda_1} \leq 1 - \frac{\beta_1}{\beta_2} = \eta^c$$

↳ heat engine regime $\rightarrow W_{\text{out}} \geq 0, Q_1 \geq 0 \rightarrow \frac{\lambda_2}{\lambda_1} > \frac{\beta_1}{\beta_2} = \frac{T_2}{T_1}$

N identical quantum Otto engines in parallel.

$$H = \sum H_i \quad H_i = \lambda K$$

$$W_{\text{out}} = N (\lambda_2 - \lambda_1) \left[U_k [P_2 \lambda_2] - U_k (P_1 \lambda_1) \right]$$



$$W_{\text{out}} \sim N$$

$$\Delta\eta \sim 1$$

$$\bar{T} = \frac{W_{\text{out}}}{\Delta\eta} \sim N$$

$$W_{\text{out}} = w_0$$

$$\Delta\eta \sim \frac{1}{N}$$

$$\bar{T} = \frac{W_{\text{out}}}{\Delta\eta} \sim N$$

In general

$$\Pi \sim \left. \frac{\partial W_{\text{out}}}{\partial \Delta \eta} \right|_{\Delta \eta=0}$$

(provided linear response applies)

Straightforward calculation

$$\left. \frac{\partial W_{\text{out}}}{\partial \Delta \eta} \right|_{\Delta \eta=0} = C_K(\alpha_1, \beta_1) = N c_k(\alpha_1, \beta_1)$$

Heat capacity

Specific heat

$$C_K = -\beta^2 \frac{\partial U_K}{\partial \theta}$$

Boost of performance Π

if heat capacity C_k scales
more than linearly



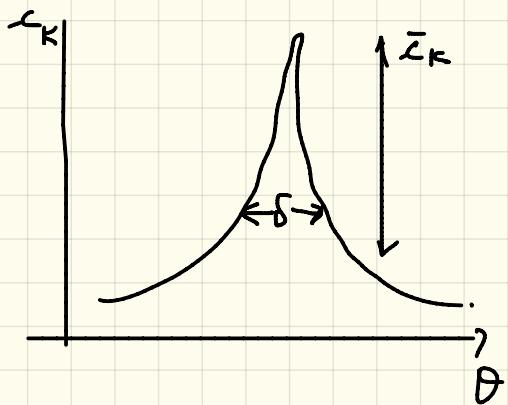
Thermodynamic limit violated

e.g. ① The system is at a phase transition

of 1st or 2nd order

② The system is "small,"

2nd order phase transition



$$\bar{C}_K \sim N^{\frac{\alpha}{d\nu}}$$

$$\delta \sim N^{-\frac{1}{d\nu}}$$

$\alpha \rightarrow$ Specific heat critical exponent

$\nu \rightarrow$ Correlation length critical exponent

$d \rightarrow$ # of dimensions

$$\Rightarrow \bar{\tau} \sim N^{1+d/\nu}$$

$$P = \frac{W_{out}}{T} \Rightarrow \dot{\tau} = \frac{\Pi}{T}$$

$$\tau \sim \text{relaxation} \sim N^{\frac{z}{d}}$$

to equilibrium

$$\Rightarrow \dot{\tau} \sim N^{1+(\alpha-2\nu)/\nu}$$

$z \rightarrow$ Dynamical critical exponent

2ND order phase transition

$$\dot{\pi} \sim N^{1 + (\alpha - z\nu)/\nu}$$

Boost if $d - z\nu > 0$

$$\Rightarrow \theta \sim N$$

$$\Delta \eta \sim \frac{1}{N^{(\alpha - z\nu)/\nu}}$$

If asymptotically occurs if

$$\alpha - z\nu \geq 1$$

$$(\delta \sim \frac{1}{N^{\alpha\nu}})$$

$$\alpha - zv > 0$$

← transient

$$\alpha - zv \geq 1$$

← Asymptotic

Possible ?

Ising 3D

$$\alpha \approx 0.12$$

$$\beta \approx 0.63$$

$$\gamma \approx 2.35$$

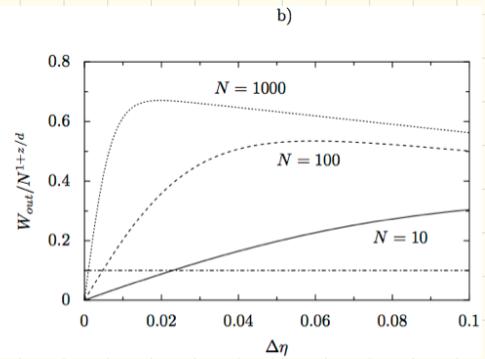
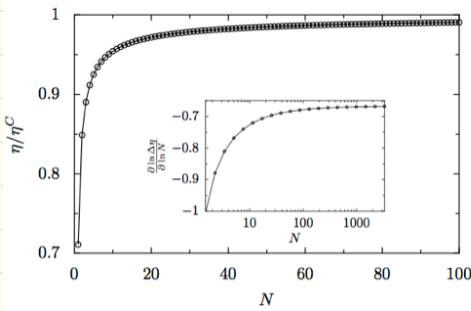
$$\Rightarrow \alpha - z\beta \approx -0.28$$



$z\nu \approx -0.7 \leftarrow$ Grams et al., Nat. Comm. 5 4853 (2014)

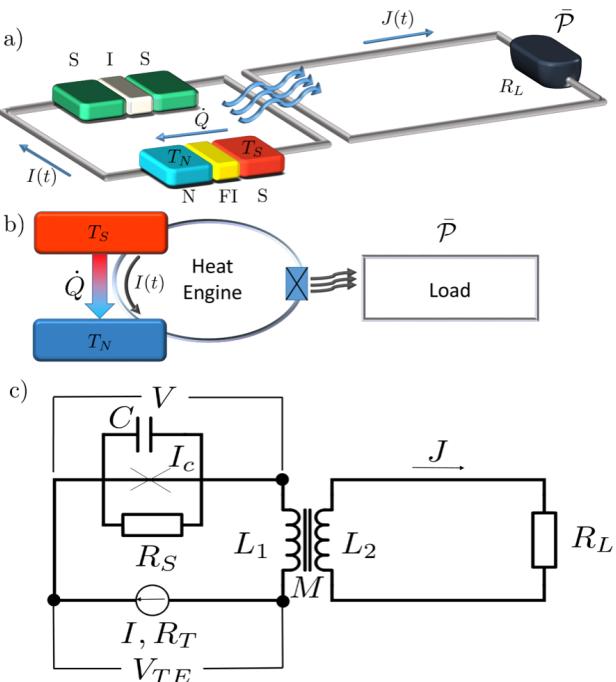
$\alpha \approx 0.38 \leftarrow$ Higashitaka et al., J Phys Soc Jpn 73 2845 (2004)

$\Rightarrow \alpha - z\nu \gtrsim 1.08$

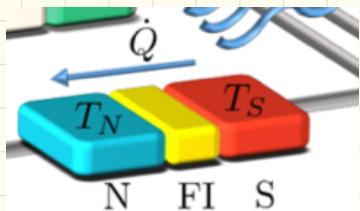


Josephson quantum heat engine

G. Marchegiani, P. Virtanen, F. Fazio, M. Gaspini, Phys. Rev. Applied 6 054014 (2016)



The N -FI-S element



$$N_{\uparrow,\downarrow}(E) = \frac{1}{2} \left| \text{Re} \left[\frac{E + i\Gamma \pm h_{exc}}{\sqrt{(E + i\Gamma \pm h_{exc})^2 - \Delta^2}} \right] \right|$$

$$I_{TE}(V_{TE}) = \frac{1}{eR_T} \int_{-\infty}^{+\infty} dE \mathcal{N}(E) [f_S(T_S) - f_N(V_{TE}, T_N)]$$

$$\mathcal{N} = N_+ + PN_-$$

$$P = (G_\uparrow - G_\downarrow)/(G_\uparrow + G_\downarrow) \quad N_\pm = N_\uparrow \pm N_\downarrow$$

$$f_N(V_{TE}, T_N) = [1 + \exp(E + eV_{TE}/k_B T_N)]^{-1}$$

$$f_S(T_S) = [1 + \exp(E/k_B T_S)]^{-1}$$

Giazotto et al

Phys. Rev. Applied 4 044016 (2015)

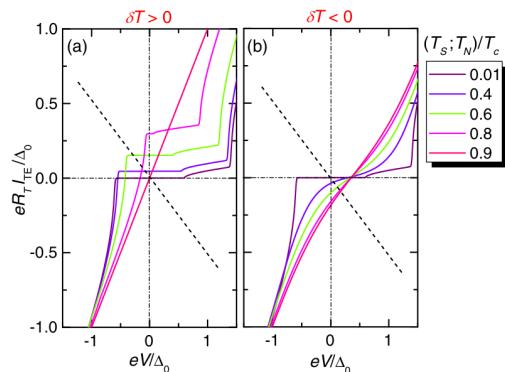


FIG. 3. Characterization of the TE for $T_S \neq T_N$. (a) I_{TE} - V characteristic of TE calculated for several values of T_S (see top legend) at $T_N = 0.01T_c$, $P = 0.9$, and $h_{exc} = 0.4\Delta_0$. (b) The same as in panel (a) calculated for several T_N values at $T_S = 0.01T_c$. Dashed lines in panels (a) and (b) represent the current (I_{JJ}) flowing through the Josephson element when it is operated in the resistive regime, $I_{JJ} = -V/R_{JJ}$.

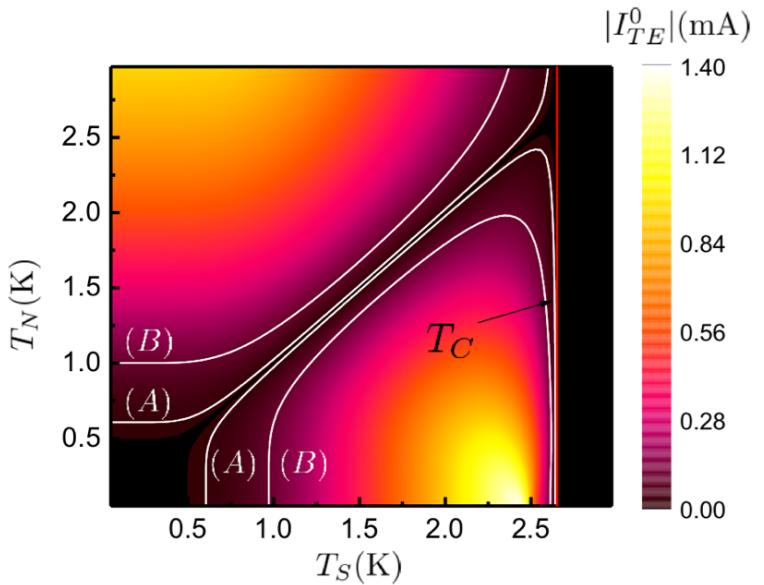
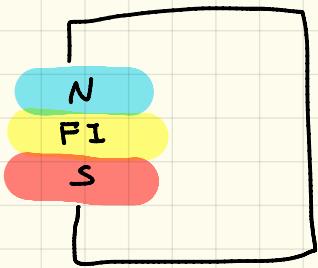


FIG. 2. (color online). Density plot of zero voltage thermoelectric current provided by the N - FI - S element. Parameters are $P = 0.9$, $T_{C0} = 3.0$ K, $R_T = 0.1$ Ω , $\Gamma = 10^{-4}$ Δ_0 and $h_{exc} = 0.4 \Delta_0$, where $\Delta_0 = 1.764 k_B T_{C0}$ and T_{C0} is the zero-field critical temperature. In the black region $|I_{TE}^0| < 3 \mu\text{A}$. The contour lines for $|I_{TE}^0| = 10 \mu\text{A}$ (A) and $|I_{TE}^0| = 100 \mu\text{A}$ (B) are plotted in white.

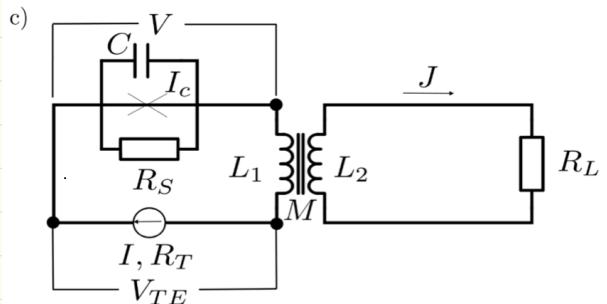
Circuit equations

$$1) \quad I = I_c \sin \varphi + \frac{V}{R_s} + C \dot{V}$$

$$2) \quad \dot{\varphi} = \frac{2\pi}{\Phi_0} V$$

$$3) \quad V_{TE} = L_1 \dot{I} - M \dot{J} + V$$

$$4) \quad M \dot{J} = L_2 \dot{J} + R_L J$$



Ansatz 2: $\bar{V}_{TE}(t) = \bar{V}_{TE} + \delta V_{TE}(t) \quad \langle \delta V_{TE} \rangle = 0$

Approx: $\bar{Z}(w) \approx \bar{Z}(0) = R_T$

$$5) \quad I(t) = -I_{TE}(\bar{V}_{TE}) - \frac{\delta V_{TE}(t)}{R_T}$$

Unknowns:

$$I, J, \varphi, \delta V_{TE}, V$$

$\bar{V}_{TE} \rightarrow$ Tuned to the point where $\langle \delta V_{TE} \rangle = 0$

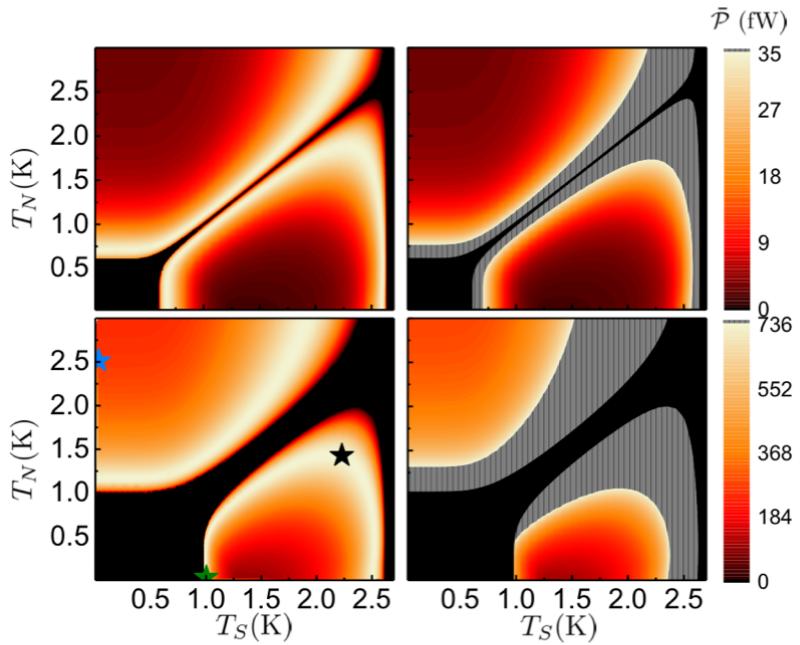


FIG. 3. (color online). Transmitted power vs thermal gradient for $I_c = 10 \mu\text{A}$ (top) and $I_c = 100 \mu\text{A}$ (bottom). Numerical data (left panels) are compared to the expression Eq. (9) (right panels). The regions where Eq. (9) gives a higher estimate than the largest value of $\bar{\mathcal{P}}$ in the corresponding numerical graph are drawn in grey. The stars denote the points investigated in Fig. 4. Parameters are $C = 100 \text{ fF}$, $L_1 = L_2 = 100 \text{ pH}$, $M = 10 \text{ pH}$, $R_S = 1 \Omega$ and $R_L = 10 \Omega$.

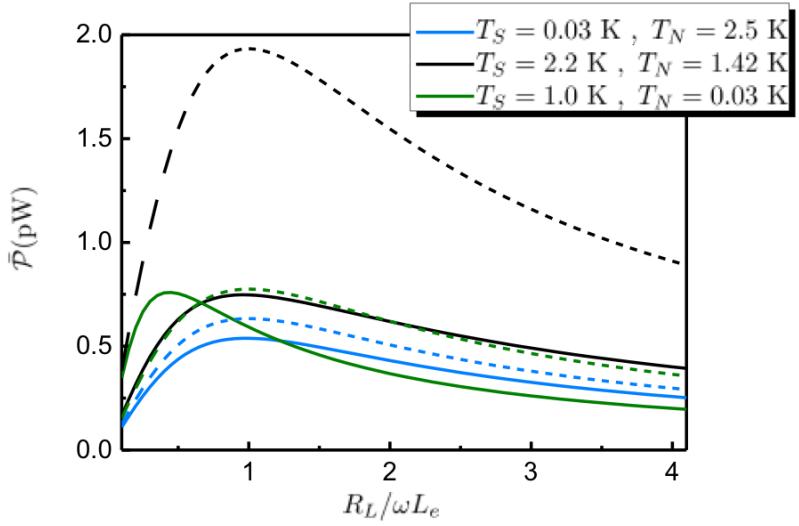


FIG. 4. (color online). Transmitted power vs load resistance for different working point in the (T_S, T_N) space, marked by the stars in Fig. 3. Numerical data (solid lines) are compared with the expression Eq.(9) (dash lines). Parameters are $C = 100$ fF, $L_1 = L_2 = 100$ pH, $M = 10$ pH, $R_S = 1$ Ω , and $I_c = 100$ μ A.